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Improving the Dynamic Accuracy of Elastic Industrial Robot Joint by Algebraic Identification Approach

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Abstract—In this paper, an improvement of the dynamic accuracy of a flexible robot joint is addressed. Based on the observation of the measured axis deformation, a simplified elastic joint model is deduced. In the first step, the non-linear model component's is analyzed and identified in the cases of the gravity bias and the friction term. In the second step, a non-asymptotically algebraic fast identification of the oscillatory behavior of the robot axis is introduced. Finally, the performances of the identification approach are exploited in order to improve the dynamic accuracy of a flexible robot axis. This is done experimentally by the combination of the adaptation of the jerk time profile to reduce the end-point vibration and the model-based pre-compensation of the end-point tracking error.

1. INTRODUCTION

Anthropomorphous robots are widely used in many fields of industry to carry out repetitive tasks such as pick and place, welding, assembling, and so on. Due to their flexibility and ability to perform complex tasks in a large workspace, robots are finding their way to realize continuous operations and especially machining applications like, prototyping, cleaning and pre-machining of cast parts ([4], [16]). Also, a high level precision end-effector trajectory is required in some continuous process. However, the static and dynamic precision are mainly limited by the robot serial structure and the joint stiffness which can induce vibrations of the end effector [13].

In order to control robot motions, we need accurate dynamical models including vibratory phenomena and most of dominating behaviors. These models are function of the geometric parameters, inertial terms and drive chain parameters such as stiffness, viscosity [18], friction and kinematic error issued from the gearbox, ([8], [19]).

In this paper we present the identification of a robot axis model taking into account of dominating phenomena [20]. Both of pre-sliding and sliding frictions by means of the Stribeck model are addressed, ([1], [14]). Moreover, the oscillatory behavior is identified based on algebraic technique introduced by Fliess and Sira-Ramirez ([5], [6]). The estimation procedure, given by exact formulas in the time domain, leads to a non asymptotic convergence. The identification algorithm is deterministic where the algebraic expressions are written as a function of integrals of system inputs and outputs [11]. In a previous study [15], by comparing this method to classical recursive approach ([9], [10]), we show its robustness and admits low complexity parameters computation efficiency when measurements are corrupted by noises.

The outline of this paper is as follows. Section 2 describes the robot system in which an elastic joint robot model is addressed in relation with a physical observation of the axis deformation. In section 3, a non linear terms derived from the gravity impact and the joint friction are identified. Here, a new reduce order model is proposed. An on-line estimation of vibratory frequency induced by

the joint flexibility and viscosity is developed in section 4. Section 5 devoted the robot axis pre-compensation scheme where experimental results are presented, while Section 6 concludes the paper.

2. ROBOTIC SYSTEM DESCRIPTION

The robotic manipulator Stäubli RX-170B is a 6 degree of freedom (DOF) industrial robot with revolute joints. The Stäubli CS8 controller provides the actual joint positions and velocities internally measured through encoders on the motor shafts. The actuator torque is measured through the motor current. The control signals and the state variables are accessible within a sampling time of 4 ms. In order to measure the real position of robot axis, a high-precision API T3 Laser Tracker is used where a Spherically Mounted Retroreflector (SMR) is mounted on the link and guaranteeing an online path tracking. The system resolution is $0.1 \mu\text{m/m}$ and the angular resolution is about $\pm 0.018 \text{ arcsec}$. A well dynamic measurement is ensured thanks to the maximum Tracker acceleration ($>2 g$). In the presented paper, subsequent modeling and identification analysis are restricted to the second robot axis which is the subject to largest variation of the load with maximum gravitational torque of almost 2150 Nm. The robot arms performs trajectories around the 2^{nd} axis with respect to its workspace, while others joints i.e. 1, 3, 4, 5 and 6 are blocked respectively to $[-90, 0, 0, 0, 0]$ deg.

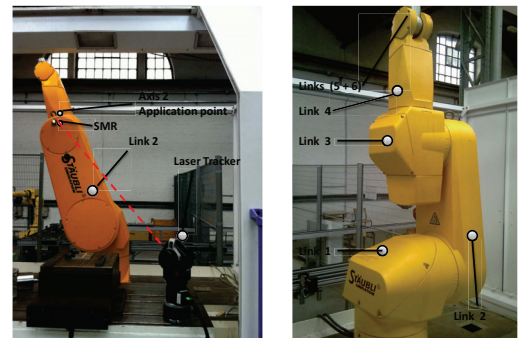


Fig. 1. Stäubli RX-170B robot arm

A. Elastic-joint robot modeling

1) *Dynamic model*: The second robot axis can be considered as a two mass coupled system, having the rigid link driven by electrical motor through a rotational joint transmission undergoing a visco-elastic deformation with a constant stiffness K and a viscosity D , [17] (see Fig. 2). The flexible joint dynamic is modeled as follows

$$J\ddot{\theta}_l(t) + D\dot{\theta}_l(t) + K\theta_l(t) = K\theta_r(t) - \tau_r(\theta_r(t)) - \tau_f(\dot{\theta}_r(t)) \quad (1)$$

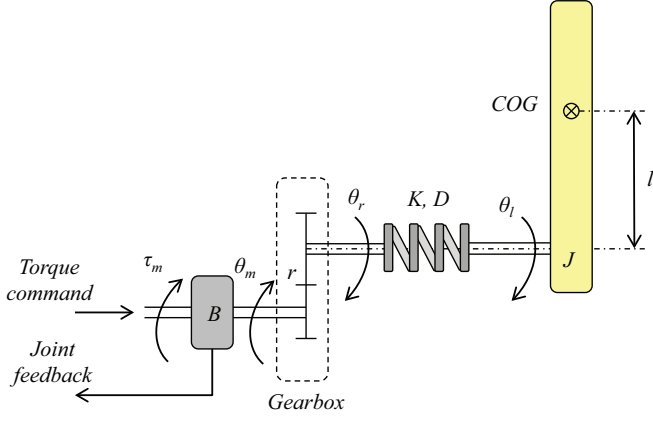


Fig. 2. Elastic robot joint

$$B\ddot{\theta}_m(t) - r^{-1}D\dot{\theta}_l(t) - r^{-1}K(\theta_l - \theta_r)(t) + r^{-1}\tau_r(\theta_r(t)) + r^{-1}\tau_f(\dot{\theta}_r(t)) = \tau_m(t) \quad (2)$$

where θ_l , $\dot{\theta}_l$ and $\ddot{\theta}_l$ (respectively θ_m , $\dot{\theta}_m$ and $\ddot{\theta}_m$) are the link (respectively the motor) positions, velocities and accelerations. θ_r is the motor position reflected through the second axis gear ratio r with $\theta_r = r \theta_m$. r is given by the Stäubli RX-170B datasheet and is equal to 183.03. J denotes the angular mass of the second robot axis under the configuration (see Fig. 1). The actuator inertia is given by B . The residual torque defined by (4) is given by $\tau_r(\theta_r)$ while $\tau_f(\dot{\theta}_r)$ depicts the friction torque defines by the motion of all axes mounted downstream of the second one. Referring to the right-hand side of the equation (2), τ_m denotes the torques supplied by the second axis motor. The motor torque τ_m is determined through the current measurement i :

$$\tau_m(t) = k_m i(t), \quad k_m = 0.51 \text{ N.m/rad} \quad (3)$$

2) *Axis deformation*: The dynamical model (1),(2) is justified by the following experimental studies. Hence, to capture the joint torsion, the robot axis tracks a position trajectory from -70 deg to 50 deg with successive constant velocities steps as depicted in Fig. 3.

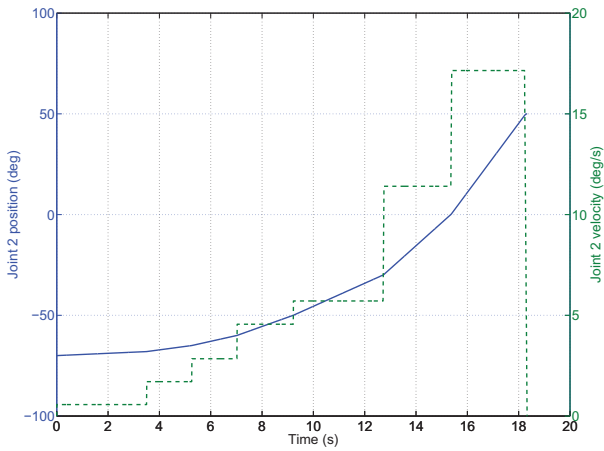


Fig. 3. Joint position reference θ_r and joint velocity reference $\dot{\theta}_r$

Fig. 4 depicts the measured tracking errors for θ_r position and the axis deformation $\Delta\theta(t) = (\theta_l(t) - \theta_r(t))$. One notes that the tracking performance of the motor is good despite the presence of acceleration discontinuities. The curves corresponding to the axis deformation

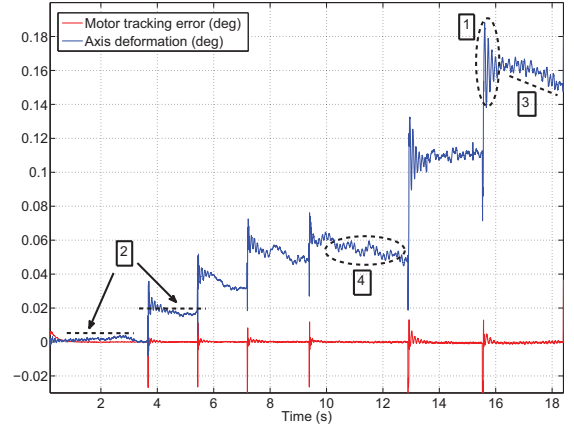


Fig. 4. Measured joint deformation $\Delta\theta$ and joint feedback position error θ_r

show four dominating phenomena which justify the proposed model. By order of contribution, each level can be explained as follows :

- 1) An oscillatory behavior at each velocity changes due to the joint stiffness K , the damping D and the moment of inertia J . It leads to a low frequency component around 10 Hz.
- 2) A static error at constant axis velocity dealing with the presence of a viscous friction term in the elastic-joint.
- 3) A time varying error at constant velocity mainly due to the residual torque.
- 4) A high frequency component of the axis deformation which correspond to the first harmonic of the gear error.

Consequently, in the next section we are interested in the parameters estimation for the three first phenomena. The gear error model is not developed in this paper.

3. REDUCE ORDER MODEL FOR ELASTIC-JOINT ROBOT AXIS

The elastic robot joint behavior results from the superposition of multiple partially coupled phenomena : friction, pre-sliding, gravity and the joint stiffness. Therefore, the experimental setup is designed such that to isolate and decouple these effects for the purpose of parameters estimation.

A. Gravitational-Balancing torques calculation

The fact that second robot axis is driven by both of actuator and balancing system torques (see Fig. 5), the so-called residual torque emerges from the superposition of previous torques depending on joint position and load mass. This can be expressed as follows:

$$\tau_r = \tau_g - \tau_b \quad (4)$$

where, τ_g is the joint gravitational torque, τ_b the torque due to the balancing system and τ_r is the residual torque. The gravitational torque is given by

$$\tau_g(\theta_r) = M_{eq} \times g \times l \times \sin(\theta_r) \quad (5)$$

where g is the gravitational acceleration. M_{eq} is the sum of masses of each arms making up the robotic system starting from the axis 2. l matches the radius vector to the gravity center of the total mass from the joint 2 origin as depicted in Fig. 2. The balancing torque τ_b is generated by the action of springs during the rotation of joint 2 (see Fig. 5) and is given by

$$\tau_b(\theta_r) = (k_b \times x(\theta_r) + P_c) \times l \quad (6)$$

in which k_b denotes the spring stiffness, $x(\theta_r)$ is the spring displacement in function of the joint position which is nonlinear and P_c is the spring preload. These parameters are obtained from robot datasheet's.

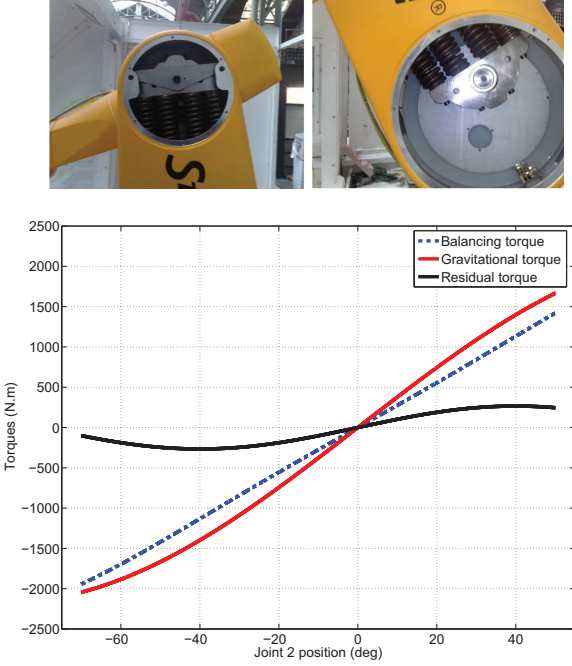


Fig. 5. Balancing system (upper) and Gravitational, Balancing and Residual torques (below)

B. Friction model and Pre-sliding friction estimation

1) *Friction model*: Aiming that gravity impact will not affect joint motion, the link is operated in the neighborhood of the residual torque free position $\theta_r \approx 0$ deg (see Fig. 5). The joint is driven back and fourth at constant velocities $[\pm 0.004 \pm 0.006, \dots, \pm 0.2]$ rad/s in a bidirectional drive experiment. Friction torque values correspond to residual joint torque subtracted from applied motor torque seen after the joint transmission r . The joint friction torque $\tau_f(\dot{\theta}_r)$ is fitted by means of the Stribeck model [3] in the sliding regime (Fig. 6). The model for friction is given by

$$\tau_f(\dot{\theta}_r) = \left(f_c + (f_s - f_c) e^{-\left| \frac{\dot{\theta}_r}{\dot{\theta}_{rs}} \right|^2} \right) \text{sgn}(\dot{\theta}_r) + f_v \dot{\theta}_r \quad (7)$$

It must be noted that Stribeck characteristic curves i.e negative and positive velocities, are fitted to the experimental data with a Gauss-Newton algorithm that minimizes the summed squares error by the instruction *lsqcurvefit* in the Matlab package. As it is shown in Fig. 6, friction torque was noticed to be non-symmetric at positive and negative velocities as it is depicted in TABLE I.

2) *Pre-sliding friction estimation*: The static friction (pre-sliding regime) is identified by means of an experiment with a constant very low velocity of $5.931 \cdot 10^{-5}$ rad/s. During the process, the velocity reference causes in the contacting surfaces of the joint transmission a cyclic motion in function of the joint displacement at the initialization of the axis movement, as shown in Fig. 7. Hence, the pre-sliding friction involved in the Stribeck model (7), is well identified and verified thanks to the experimental identification and the curve fitting algorithm. It must be noted that according to the estimation algorithm, $F_{sModel} = 148.5443$ Nm and by referring to the experimental pre-sliding test, $F_{sExper} \approx 150$ Nm.

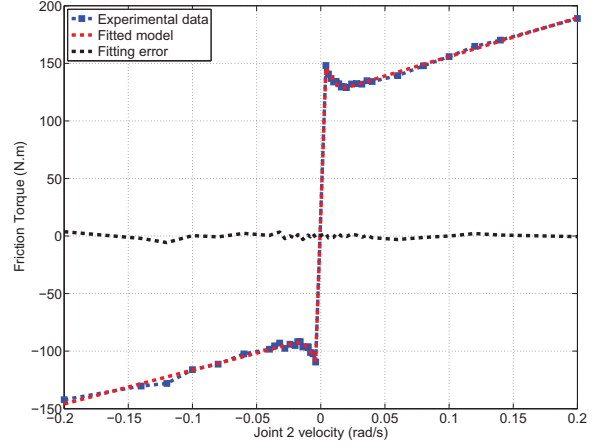


Fig. 6. Experimental friction torque and fitted Stribeck curve

TABLE I
STRIBECK FRICTION COEFFICIENTS

Joint 2 positive velocity			
$f_c = 122.19$ Nm	$f_s = 148.54$ Nm	$f_v = 336.73$ Nm.s/rad	$\dot{\theta}_{rs} = 0.009$ rad/s
Joint 2 negative velocity			
$f_c = 87.02$ Nm	$f_s = 111.38$ Nm	$f_v = 294.44$ Nm.s/rad	$\dot{\theta}_{rs} = 0.009$ rad/s

C. Angular mass, Stiffness and Damping estimation

In this subsection we are interested in the estimation of the moment of inertia J , the axis stiffness K and the damping coefficient D . The identification procedure are performed with respect to the previous robot configuration where joints 1, 3, 4, 5 and 6 are fixed respectively to $[-90, 0, 0, 0, 0]$ deg.

1) *Angular mass identification*: The estimation procedure consists in performing a constant acceleration profile where the robot axis tracks a first order velocity polynomial's. Assuming that rotor inertia B in (2) is obtained from the manufacturer data, J is then deduced by means of the division of acting torques in the joint side through the acceleration vector. In Fig. 8, experimental result of J identification is revealed where the inertial torque is computed from the sum of captured joint friction (6), gravitational impact (4) subtracted from the acting motor torque.

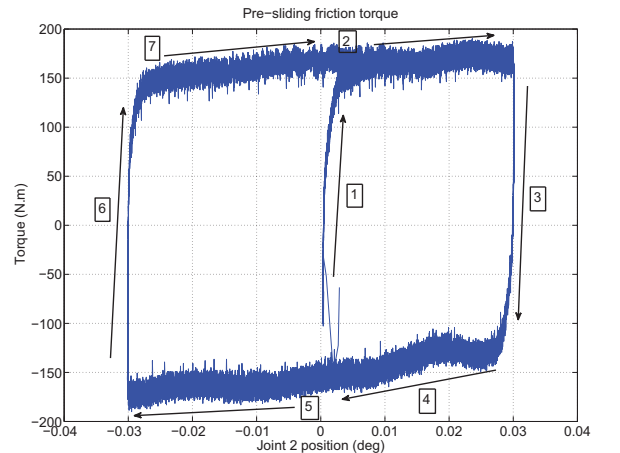


Fig. 7. Pre-sliding friction at $\theta_r = 0$ deg

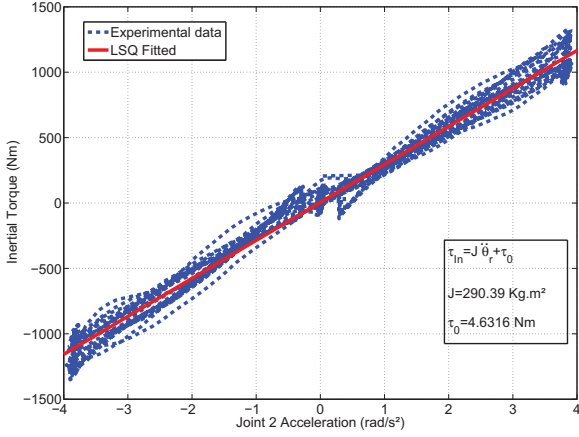


Fig. 8. Joint 2 moment of inertia : experimental data and fitted model

2) *Stiffness and damping estimation*: Focus on joint torsion (Fig. 9), a second order model can be locally identified where the contributed parameters are limited to J , K and D . An estimation of

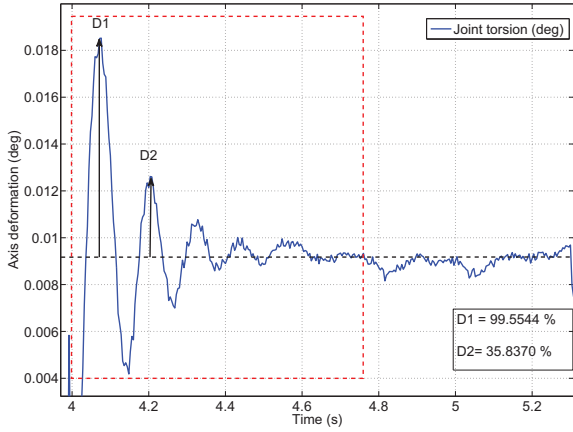


Fig. 9. Robot axis deformation

the stiffness and the damping coefficients is performed based on the computation of the logarithmic decrement δ , damping ratio ζ and the vibration period T .

$$\begin{cases} \zeta = 0.1626 \\ \omega_0 = 60.968 \text{ rad/s} \end{cases} \Rightarrow \begin{cases} K = J\omega_0^2 = 1.0794 \cdot 10^6 \text{ N.m/rad} \\ D = 2\zeta\sqrt{KJ} = 5.7575 \cdot 10^3 \text{ N.m.s/rad} \end{cases} \quad (8)$$

D. Reduce order model

One notes that the motor tracking error does not exceed 0.5 % of the reference position signal (Fig. 4). Indeed, we can conclude that the controller tracking error can be neglected as compared to the end-point tracking error i.e the joint deformation $\Delta\theta$. This realistic hypothesis conducts to a reduce order model which can be limited to (1). The input data is then the motor position reflected through the nominal value of the gear ratio r i.e $r\theta_m$ measured with the robot controller and the output is the link position θ_l measured with the laser tracker device. Fig. 10 depicts the model validation scheme.

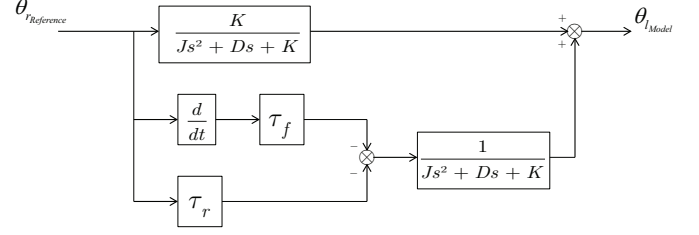


Fig. 10. Validation block diagram

4. NON-ASYMPTOTIC METHOD FOR REAL-TIME ESTIMATION OF VIBRATION FREQUENCY

In this section, we provide the reader with the estimation of the parameters involved in the second order model of the robot axis, i.e. the damping ratio ζ and the resonance frequency ω_0 . It allows us to validate the physical identification of the involved coefficients in the previous subsection. The non-asymptotic identification of these parameters allows the user to capture the axis' dynamics for each time step on the one hand and to design a feed-forward control to compensate the end-point tracking error on the other hand. Moreover, the real-time ability of the estimation algorithm is promising for counteract the time variations of parameters in robotic applications.

A. Method description

Since the main contribution of the axis's dynamics is issued from its oscillatory behavior (damping oscillations Fig. 9), it will be shown that the two parameters ζ and ω_0 can be estimated under the transient response of the joint deformation using equation (9).

$$J\ddot{\theta}_l(t) + D\dot{\theta}_l(t) + K\theta_l(t) = K\theta_r(t) \quad (9)$$

Assuming that ζ is the damping ratio and ω_0 is the resonance frequency, (9) can be rewritten as

$$\ddot{\theta}_l(t) = -\lambda_1\dot{\theta}_l(t) - \lambda_2\theta_l(t) + \lambda_2\theta_r(t) \quad (10)$$

where $\lambda_1 = 2\zeta\omega_0$ and $\lambda_2 = \omega_0^2$. Indeed, using an algebraic approach for (10), it is possible to express the parameters λ_1 and λ_2 as a function of the measured input θ_r and the output θ_l .

Assuming that $\hat{\theta}_l(s)$ and $\hat{\theta}_r(s)$ are the Laplace transform of $\theta_l(t)$ and $\theta_r(t)$, we consider the Laplace transform of (10)

$$s^2\hat{\theta}_l(s) - s\theta_l(0) - \dot{\theta}_l(0) = -\lambda_1(s\hat{\theta}_l(s) - \theta_l(0)) - \lambda_2(\hat{\theta}_l(s) - \hat{\theta}_r(s)) \quad (11)$$

In order to eliminate the initial condition, we apply the derivative operator with respect to s two times. It leads to

$$s^2\frac{d^2\hat{\theta}_l(s)}{ds^2} + 4s\frac{d\hat{\theta}_l(s)}{ds} + 2\hat{\theta}_l(s) = \lambda_1\left(-s\frac{d^2\hat{\theta}_l(s)}{ds^2} - 2\frac{d\hat{\theta}_l(s)}{ds}\right) + \lambda_2\left(\frac{d^2\hat{\theta}_l(s)}{ds^2} - \frac{d^2\hat{\theta}_r(s)}{ds^2}\right) \quad (12)$$

It remind that differentiation with respect to s in the operational domain results in a multiplication by $-t$ in the time domain. The multiplication by s in the operational domain leads to the derivation in the time domain. As it well known that derivation amplify the high frequency and then the noise contribution, so the application of the estimator (12) is not convenient. Thus, we divide (12) by s^3 in order to cancel the derivation terms and obtain a relationship with integral operators. Consequently, applying the classical cauchy formula, we compute the inverse Laplace transform and return to time domain as follows:

$$\mathcal{L}^{-1}\left(\frac{1}{s^m}\frac{d^n\theta(s)}{ds^n}\right) = \frac{(-1)^n}{(m-1)!} \int_0^t (t-\tau)^{m-1} t^n \theta(\tau) d\tau = P_{m,n}^\theta(t)$$

It yields,

$$P_{1,2}^{\theta_l}(t) + 4P_{2,1}^{\theta_l}(t) + 2P_{3,0}^{\theta_l}(t) = \lambda_1(-P_{2,2}^{\theta_l}(t) - 2P_{3,1}^{\theta_l}(t)) + \lambda_2(P_{3,2}^{\theta_l}(t) - P_{3,2}^{\theta_r}(t)) \quad (13)$$

Then, we have to generate one more equation in order to obtain a linear system so as to identify λ_1 and λ_2 . It is well known that integration in time domain is a linear operation, thus, by taking (12) and dividing it one time with respect to s^4 , one gets the following linear system in the time domain

$$P_{2,2}^{\theta_l}(t) + 4P_{3,1}^{\theta_l}(t) + 2P_{4,0}^{\theta_l}(t) = \lambda_1(-P_{3,2}^{\theta_l}(t) - 2P_{4,1}^{\theta_l}(t)) + \lambda_2(P_{4,2}^{\theta_l}(t) - P_{4,2}^{\theta_r}(t)) \quad (14)$$

Equations (13) and (14) allow to obtain expressions of the estimate of λ_1 and λ_2 .

$$P(t)\Theta = Q(t) \quad (15)$$

$$\text{where } \Theta = [\lambda_1 \ \lambda_2]^T, P(t) = \begin{pmatrix} -P_{2,2}^{\theta_l}(t) - 2P_{3,1}^{\theta_l}(t) & P_{3,2}^{\theta_l}(t) - P_{3,2}^{\theta_r}(t) \\ -P_{3,2}^{\theta_l}(t) - 2P_{4,1}^{\theta_l}(t) & P_{4,2}^{\theta_l}(t) - P_{4,2}^{\theta_r}(t) \end{pmatrix}$$

$$\text{and, } Q(t) = \begin{pmatrix} P_{1,2}^{\theta_l}(t) + 4P_{2,1}^{\theta_l}(t) + 2P_{3,0}^{\theta_l}(t) \\ P_{2,2}^{\theta_l}(t) + 4P_{3,1}^{\theta_l}(t) + 2P_{4,0}^{\theta_l}(t) \end{pmatrix}$$

For a given value $t > 0$ and if $P(t)$ is invertible, then, an estimation of Θ is obtained by

$$\begin{pmatrix} \lambda_1 \\ \lambda_2 \end{pmatrix} = P(t)^{-1}Q(t) \quad (16)$$

Note that (13) and (14) are expressed in terms of a linear combination of iterated convolution integrals over process input and output. However, there is two unknown parameters to estimate.

B. Implementation

Aiming to estimate the unknown parameters, the algebraic estimator i.e the iterated integrals present in (13) and (14) have been discretized using the trapezoidal rule [2]. Besides, in order to estimate $\hat{\lambda}_1$ and $\hat{\lambda}_2$ not asymptotically, integrating over a short time interval $[0, t]$ provides an online parametric estimator. Meanwhile, t cannot obviously be taken arbitrary small even in a noise free context [15]. A lower bound for t has been formally characterized in [7], within the framework of nonstandard analysis. Let us quote that the estimation time T_w may be small, resulting in fast estimation. The experience shows that a sliding window $t = 20 T_s$ with T_s be the sampling time equals to 0.004 s gives good results. From the experimental

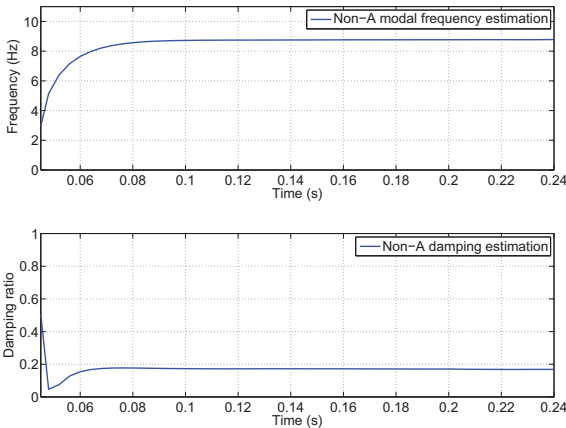


Fig. 11. Experimental estimation of the parameters

data, the experience shows that after 0.08 seconds the parameters have converged. Besides, one gets an estimation of $\hat{f}_0 = \frac{\omega_0}{2\pi} = 8.972$ hz and $\hat{\zeta} = 0.1722$. Indeed, when they are compared to the ones obtained by physical identification method (see section 3.C). We have an estimation error less than 7.35 % for ω_0 and about 5.9 % for ζ .

C. Model validation

From the protocol test defined in Fig. 3, we depict in Fig. 12 the comparison between the measured joint deformation and the simulated one through the identified model response. The parametric identification via the non-asymptotic approach for the linear part of the reduce order model (ω_0 , ζ) and the non-linear contribution derived from the gravity impact and the friction term, converges as well as possible to the real process behavior. Furthermore, the model presents a good fit equal to 82 %. The modeling error can be explained through some uncertainties in the estimation of the friction and the calculation of the residual torques where the balancing torque include the pre-load variation. In addition, the performances of the

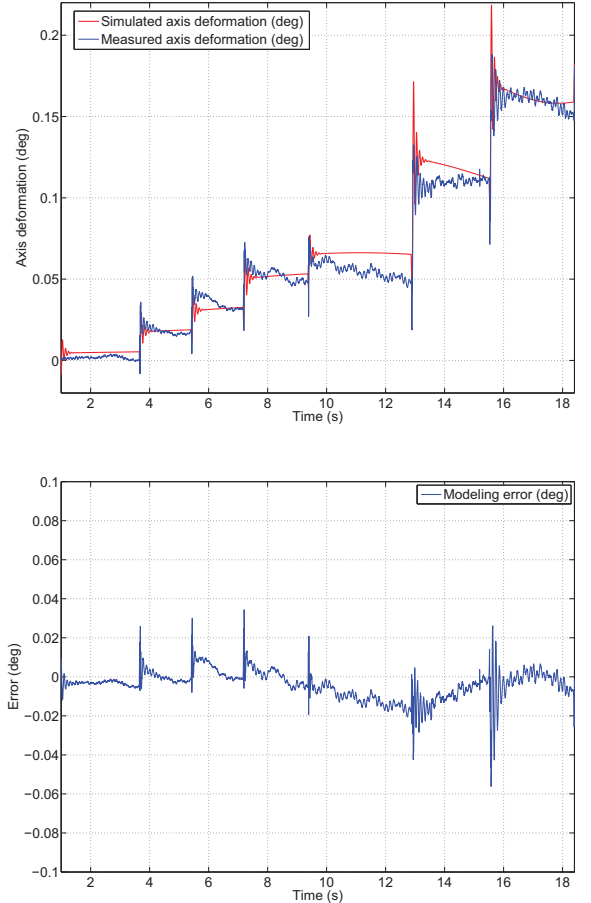


Fig. 12. Axis deformation model validation (upper) and modeling error (below)

estimation's algorithm i.e robustness and convergence rapidity can be strongly exploited so as to improve the dynamic accuracy of the robot axis.

5. TOWARDS IMPROVING THE DYNAMIC ACCURACY OF THE ROBOT AXIS VIA TRAJECTORY PRE-COMPENSATION

In order to reduce the second robot axis deformation, the strategy of joint pre-compensation consists in designing a feed-forward

control so as to anticipate the end-point tracking error based on the identified model:

$$\theta_{m_command}(t) = \theta_{m_reference}(t) - \Delta\theta_{simulated}(t) \quad (17)$$

where, $\theta_{m_command}$ is the new joint command anticipation, $\theta_{m_reference}$ is the theoretical joint reference and $\Delta\theta_{simulated}$ the simulated joint deformation obtained from the axis model.

A. Experimental validation

The validation experiments is performed under the joint 2 position profile (Fig. 13), where the jerk profile (acceleration time derivative) is chosen so as to reduce the oscillatory behavior of the studied axis. This is done settled on the natural frequency estimation given by the identification approach (section 4). For further details about jerk time methodology, see [13]. Fig. 14 depicts the evolution of

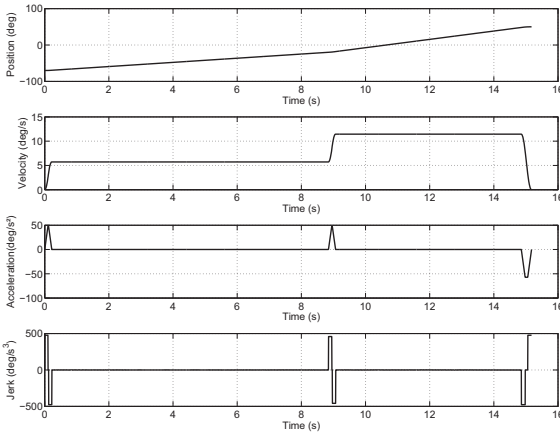


Fig. 13. Joint 2 position profile and its time derivatives resulting from path planning algorithm

the joint deformation before and after trajectory compensation. One can conclude that the amplitude of $\Delta\theta$ is at least two times smaller after trajectory anticipation and leads to a significant reduction of the end-point tracking error. We must take into consideration that the compensation law does not integrate all the error sources, such as the kinematic error of the gear [8], [19], which will be investigated and integrated in a future work.

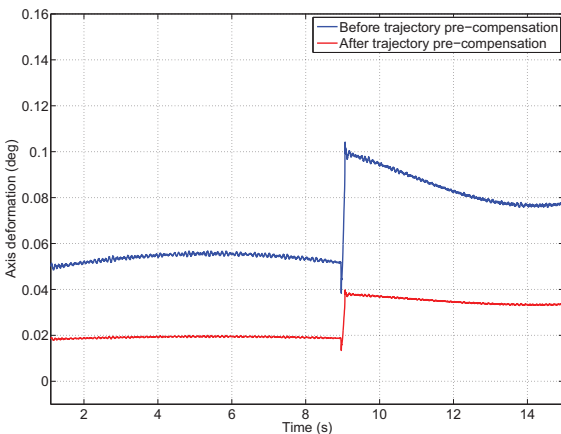


Fig. 14. $\Delta\theta$ without and with pre-compensation of trajectory given in Fig. 13

6. CONCLUSION

The objective of this paper was to improve the dynamic accuracy of an elastic industrial robot joint. A reduced global model composed of a linear and a non-linear part was investigated and developed based on experiments. Also, our approach provides a mean to capture the major phenomena acting on the joint deformation. The non-linear terms derived from the gravity impact and the joint friction in both pre-sliding and sliding regimes of system drive are described. It has been presented a non-asymptotic estimator to the fast and reliable identification of dynamical parameters of the linear part of the model. The estimation algorithm is able to estimate simultaneously and in a fast way (less than 80 ms) the modal frequency and the damping ratio. The computational complexity of the estimator is low, robust and the experimental results are promising. Moreover, based on the performances of the identification technique, a pre-compensation scheme of the end-point tracking error was designed. Hence, using the identified model, the experiments show that the axis trajectories are well tracked after joint position anticipation. As improvement, it will be interesting to investigate the spatial varying error due to the kinematic error of the gear and to include it into the model. Thus, the identified model and compensation strategies are useful for the control design that is applicable to all robots of similar architecture.

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